Analysis of a Bidirectional Coupled-Inductor Ćuk Converter Operating in Sliding Mode

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Abstract—Analytic models for a bidirectional coupled-inductor Ćuk converter operating in sliding mode are described. Using a linear combination of the converter four state variable errors as a general switching surface, the expression for the equivalent control is derived and the coordinates of the equilibrium point are obtained. Particular cases of the general switching surface are subsequently analyzed in detail: 1) surfaces for ideal line regulation, 2) surfaces for ideal load regulation, and 3) surfaces for hysteretic current control. Simulation results verifying the analytical predictions are presented.

Index Terms— DC-to-DC switching converters, sliding mode, variable structure systems.

I. INTRODUCTION

COUPLED-INDUCTOR Ćuk converters have been extensively used in the design of high efficiency minimum weight power processing units with extremely low conducted noise requirements [1]–[3]. Due to the significant properties of the Ćuk converter, a considerable amount of research has been devoted to the modeling, analysis, and applications of that structure [4]–[8].

However, in spite of the converter topological simplicity, the dynamic analysis constitutes a more involved problem. The controller design following the state-space averaging method [4], based on the use of a pulse width modulator (PWM) and a linear low-frequency characterization of the converter, is rather difficult in the two conduction modes. In the continuous conduction mode, the dynamic response exhibits a four order nonminimum phase behavior which requires bandwidth limitations to guarantee the regulator closed-loop stability. This requirement is accomplished by using a big capacitor to transfer the energy between input and output

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stages. Moreover, the analysis of the dynamic response is considerably complex in the discontinuous conduction mode, where the instant in which the switch current becomes zero depends on the converter state and not on its duty cycle [9].

In order to avoid the discontinuous conduction mode and, at the same time, to allow the existence of light load levels, the bidirectional switch was introduced [10], this having led to a bilinear description of the converter [16]. However, the corresponding small signal model based on the state-space averaging method cannot predict the regulator behavior when a large-signal perturbation appears.

These drawbacks have prompted some authors to explore nonlinear techniques based on the application of variable structure theory [11], [12] and the associated sliding regimes. The sliding-mode control is a time-domain technique suitable for large-signal applications exhibiting low sensitivity to external perturbations. Sliding mode analysis of the coupledinductor Cuk converter has been tangentially treated in the work of Sira-Ramirez on sliding motions in switching power converters [17]. The sliding surfaces considered are of the form $x_i = K$, for some state variable that is desired to be regulated at the level K. The restriction to this type of surface has disadvantages. Namely, only the sliding surface representing constant input current leads to an stable equilibrium point, but it does not provide output voltage regulation. Papers [18]–[21] deal specifically with the Ćuk converter operating in sliding mode. The work reported in [18] assumes that the capacitor for energy transference is considerably bigger than the output capacitor in order to decouple input and output stages. With this parametric constraint, the coupled-inductor Ćuk converter can be controlled like a buck switching cell [14], [15], but the input current exhibits an uncontrollable dynamic behavior. In another direction, [19] has considered time-varying surfaces to attenuate perturbations effects caused by input voltage or load variations. A relative insensitivity of the output voltage to such variations has been demonstrated with simulations. However, the implementation of the control algorithm is complex and requires accurate parameter and state information. The development of sliding mode control laws for a Cuk converter without magnetic coupling is the subject of the work of Malesani et al. reported in [20]. The sliding surface considered in [20] is a linear combination of the four state variable errors resulting in simple control implementation, minimum size of energy transfer capacitor, and excellent static and dynamic performances. However, there is a lack of analytic investigation on the influence of

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Fig. 1. Block diagram of a sliding-mode controlled bidirectional coupled-inductor Ćuk switching regulator.

the surface coefficients on the regulator behavior, these coefficients being eventually obtained through numerical analysis. In a more recent paper [21], a sliding mode control strategy has been proposed for a third-order Ćuk converter. Analytic expressions are given for the equilibrium points corresponding to switching surfaces that are linear combinations of up to three state variables. The corresponding experimental results show a relative insensitivity of the output voltage with respect to input voltage variations. However, the absence of the output filtering capacitor results in a significant amount of ripple in the output voltage which makes the third-order Ćuk converter inappropriate for practical purposes.

The aim of this paper is to develop analytic models for the dynamic behavior of a bidirectional coupled-inductor Ćuk converter operating in sliding mode. The approach is based on the equivalent control method [17] and uses a linear combination of the converter state variable errors as a general switching surface as shown in Section II. Section III is devoted to the study of different particular cases of the general sliding surface such as surfaces for ideal line regulation, ideal load regulation, and hysteretic current control. Section IV contains conclusions and some suggestions for further research.

II. SLIDING DOMAIN

Fig. 1 shows the block diagram of a sliding-mode controlled bidirectional coupled-inductor Ćuk switching regulator. Transistors and diodes are assumed to be ideal, and no parasitics or storage time modulation effects have been considered. In addition, losses in the passive components have been neglected.

Due to the bidirectional current capability of the switch, the discontinuous conduction mode does not take place, and therefore the converter has only one structural change during a switching period. Considering u = 1 during the switch S_1 conduction subinterval and u = 0 during the switch S_2 conduction subinterval, the converter dynamics can be described as follows:

$$\frac{di_{L1}}{dt} = -\frac{v_{C1}}{L_1} \left(1 - u\right) + \frac{V_g}{L_1} - \frac{M}{L_1} \frac{di_{L2}}{dt} \tag{1}$$

$$\frac{di_{L2}}{dt} = \frac{v_{C1}}{L_2} u + \frac{v_{Co}}{L_2} - \frac{M}{L_2} \frac{di_{L1}}{dt}$$
(2)

$$\frac{dv_{C1}}{dt} = -\frac{i_{L2}}{C_1}u + \frac{i_{L1}}{C_1}(1-u)$$
(3)
$$\frac{dv_{Co}}{dv_{Co}} = \frac{i_{L2}}{i_{L2}}v_{Co}$$
(4)

$$\frac{v_{Co}}{dt} = -\frac{i_{L2}}{C_o} - \frac{v_{Co}}{R_o C_o}.$$
 (4)

Defining the normalized variables $x_1 = \sqrt{L_1} i_1, x_2 = \sqrt{L_2} i_2, x_3 = \sqrt{C_1} v_{C_1}, x_4 = \sqrt{C_o} v_{Co}, (1)$ –(4) become

$$\dot{x}_1 = -\frac{\omega_1 x_3}{1-k^2} \left(1-u\right) + \frac{e}{1-k^2} - \frac{k\omega_{21} x_3}{1-k^2} u - \frac{k\omega_2 x_4}{1-k^2}$$
(5)

$$\dot{x}_2 = \frac{\omega_{21}x_3}{1-k^2}u + \frac{\omega_2x_4}{1-k^2} + \frac{k\omega_1x_3}{1-k^2}(1-u) - \frac{ke}{1-k^2} \quad (6)$$

$$\dot{x}_3 = -\omega_{21}x_2u + \omega_1x_1(1-u) \tag{7}$$

$$\dot{x}_4 = -\omega_2 x_2 - \omega_R x_4 \tag{8}$$

where

$$\omega_1 = \frac{1}{\sqrt{L_1 C_1}}, \ \omega_2 = \frac{1}{\sqrt{L_2 C_o}},
 \omega_{21} = \frac{1}{\sqrt{L_2 C_1}}, \ \omega_R = \frac{1}{R_o C_o},
 e = \frac{V_g}{\sqrt{L_1}}, \ k = \frac{M}{\sqrt{L_1 L_2}} \quad (-1 \le k \le 1).$$

Writing (5)-(8) in matrix form results in

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \\ \dot{x}_3 \\ \dot{x}_4 \end{bmatrix} = \begin{bmatrix} 0 & 0 & \frac{-\omega_1}{1-k^2} & \frac{-k\omega_2}{1-k^2} \\ 0 & 0 & \frac{k\omega_1}{1-k^2} & \frac{\omega_2}{1-k^2} \\ \omega_1 & 0 & 0 & 0 \\ 0 & -\omega_2 & 0 & -\omega_R \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix} + u \begin{bmatrix} 0 & 0 & \frac{\omega_1-k\omega_{21}}{1-k^2} & 0 \\ 0 & 0 & \frac{\omega_{21}-k\omega_1}{1-k^2} & 0 \\ -\omega_1 & -\omega_{21} & 0 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \\ x_3 \\ x_4 \end{bmatrix} + \begin{bmatrix} \frac{e}{1-k^2} \\ -\frac{ke}{1-k^2} \\ 0 \\ 0 \end{bmatrix}.$$
(9)

Consider now as sliding surface S(x) a linear combination of state variable errors E_i

$$S(x) = m_1 E_1 + m_2 E_2 + m_3 E_3 + m_4 E_4$$
(10)

where $E_1 = i_{L1} - i_{L1}^*$, $E_2 = i_{L2} - i_{L2}^*$, $E_3 = v_{C1} - v_{C1}^*$, $E_4 = v_{Co} - v_{Co}^*$, and $(i_{L1}^*, i_{L2}^*, v_{C1}^*, v_{Co}^*)$ are the desired values of the equilibrium point coordinates.

In terms of the normalized variables, (10) becomes

$$S(x) = \alpha x_1 + \beta x_2 + \gamma x_3 + \delta x_4 - \theta.$$
(11)

On the average, the system motion takes place on the surface S(x) = 0, whose corresponding gradient dS will be given by

$$dS = (\alpha, \beta, \gamma, \delta). \tag{12}$$

Equation (9) can also be expressed as follows:

$$\dot{x} = (Ax + D) + (Bx + C)u \tag{13}$$

where

$$A = \begin{bmatrix} 0 & 0 & \frac{-\omega_1}{1-k^2} & \frac{-k\omega_2}{1-k^2} \\ 0 & 0 & \frac{k\omega_1}{1-k^2} & \frac{\omega_2}{1-k^2} \\ \omega_1 & 0 & 0 & 0 \\ 0 & -\omega_2 & 0 & -\omega_R \end{bmatrix} \quad D = \begin{bmatrix} \frac{e}{1-k^2} \\ -\frac{ke}{1-k^2} \\ 0 \\ 0 \end{bmatrix}$$
$$B = \begin{bmatrix} 0 & 0 & \frac{\omega_1 - k\omega_{21}}{1-k^2} & 0 \\ 0 & 0 & \frac{\omega_{21} - k\omega_1}{1-k^2} & 0 \\ -\omega_1 & -\omega_{21} & 0 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix}$$

 $C = \emptyset$ (null matrix).

A necessary condition for the existence of a sliding motion in S(x) is given by the transversality condition [17]

$$\langle dS, Bx + C \rangle \neq 0 \tag{14}$$

where the notation $\langle a, b \rangle$ denotes the scalar product of a and b. From (12)–(14), we derive

$$\alpha \frac{\omega_1 - k\omega_{21}}{1 - k^2} x_3 + \beta \frac{\omega_{21} - k\omega_1}{1 - k^2} x_3 - \gamma \omega_1 x_1 - \gamma \omega_{21} x_2 \neq 0.$$
(15)

Hence, surfaces of the form $\delta x_4 = \theta$ are disregarded since they cannot lead the converter to a sliding regime.

In the method of equivalent control, the discrete function u is substituted by a continuous function $u_{eq}(x)$ which can take all the values between 0 and 1. $U_{eq}(x)$ is the equivalent control and represents the control law that describes the behavior of

the system restricted to the switching surface. The expression of $u_{eq}(x)$ [17] is given by

$$u_{\rm eq}(x) = -\frac{\langle dS, Ax + D \rangle}{\langle dS, Bx + C \rangle}.$$
 (16)

From (12)–(16), we obtain (17), shown at the bottom of the page, which is bounded by the minimum and maximum values of u

$$0 < u_{\rm eq} < 1 \tag{18}$$

thus providing the sliding domain.

Substituting u by $u_{eq}(x)$ in (5)–(8) results in the following ideal sliding dynamics:

$$\dot{x}_{1} = -\frac{\omega_{1}x_{3}}{1-k^{2}}\left(1-u_{eq}\right) + \frac{e}{1-k^{2}} - \frac{k\omega_{21}x_{3}}{1-k^{2}}u_{eq} - \frac{k\omega_{2}x_{4}}{1-k^{2}}$$
(19)
$$\dot{x}_{2} = \frac{\omega_{21}x_{3}}{1-k^{2}}u_{eq} + \frac{\omega_{2}x_{4}}{1-k^{2}} + \frac{k\omega_{1}x_{3}}{1-k^{2}}\left(1-u_{eq}\right) - \frac{ke}{1-k^{2}}$$

$$1 - k^{2} - 1 - k^{2} + 1 - k^{2} + 1 - k^{2} + 1 - k^{2} + 1 - k^{2}$$
(20)

$$\dot{x}_3 = -\omega_{21}x_2u_{\rm eq} + \omega_1x_1(1 - u_{\rm eq}) \tag{21}$$

$$\dot{x}_4 = -\omega_2 x_2 - \omega_R x_4 \tag{22}$$

where $u_{eq}(x)$ is given by (17).

The equilibrium point of (19)–(22) results from the intersection of the curve of equilibrium points and the discontinuity surface. Namely,

$$x_1^* = \frac{\omega_R}{e} \, (x_4^*)^2 \tag{23}$$

$$x_2^* = -\frac{\omega_R}{\omega_2} x_4^* \tag{24}$$

$$x_3^* = \frac{e}{\omega_1} - x_4^* \frac{\omega_2}{\omega_{21}}$$
(25)

$$\theta = \alpha x_1^* + \beta x_2^* + \gamma x_3^* + \delta x_4^*.$$
(26)

From (23)–(26) we derive

$$\frac{\alpha\omega_R}{e}x_4^{*2} + \left(\delta - \frac{\beta\omega_R}{\omega_2} - \frac{\gamma\omega_2}{\omega_{21}}\right)x_4^* + \frac{\gamma e}{\omega_1} - \theta = 0 \quad (27)$$

and therefore x_4^* will be given by (28), shown at the bottom of the page, or

$$x_{4}^{*} = \frac{\theta - \frac{\gamma e}{\omega_{1}}}{\delta - \frac{\beta \omega_{R}}{\omega_{2}} - \frac{\gamma \omega_{2}}{\omega_{21}}} \quad \text{if } \alpha = 0 \text{ and}$$
$$\delta - \frac{\beta \omega_{R}}{\omega_{2}} - \frac{\gamma \omega_{2}}{\omega_{21}} \neq 0. \tag{29}$$

$$u_{\rm eq}(x) = \frac{\alpha(\omega_1 x_3 + k\omega_2 x_4 - e) - \beta(k\omega_1 x_3 + \omega_2 x_4 - ke) + (-\gamma\omega_1 x_1 + \delta\omega_2 x_2 + \delta\omega_R x_4)(1 - k^2)}{\alpha(\omega_1 - k\omega_{21})x_3 + \beta(\omega_{21} - k\omega_1)x_3 - \gamma(1 - k^2)(\omega_1 x_1 + \omega_{21} x_2)}$$
(17)

$$x_{4}^{*} = \frac{-\left(\delta - \frac{\beta\omega_{R}}{\omega_{2}} - \frac{\gamma\omega_{2}}{\omega_{21}}\right) \pm \sqrt{\left(\delta - \frac{\beta\omega_{R}}{\omega_{2}} - \frac{\gamma\omega_{2}}{\omega_{21}}\right)^{2} - \frac{4\alpha\omega_{R}}{e}\left(\frac{\gamma e}{\omega_{1}} - \theta\right)}}{\frac{2\alpha\omega_{R}}{e}} \qquad \text{if } \alpha \neq 0 \tag{28}$$

Moreover, the expression of the equivalent control in the equilibrium point will be given by

$$u_{\rm eq}^* = \frac{1}{1 - \frac{\omega_{21}e}{\omega_1\omega_2} \frac{1}{x_1^*}}.$$
 (30)

From (18) and (30), we obtain

$$0 < -\frac{\omega_{21}e}{\omega_1\omega_2} \frac{1}{x_4^*} \le 1$$
(31)

which in terms of the physical variables of the converter can be expressed as

$$0 < -\frac{V_g}{v_{Co}^*} \ge 1. \tag{32}$$

Inequality (32) describes the sign inversion in the output voltage of the Ćuk converter and the possibility of stepdown or step-up characteristics. It also establishes under which conditions an equilibrium point belongs to the sliding domain.

III. SLIDING SURFACES

A. Sliding Surfaces of the Form $x_i = \theta$

There are three sliding surfaces of the form $x_i = \theta$ which satisfy the transversality condition. Namely,

$$x_1 = \theta \tag{33}$$

$$x_2 = \theta \tag{34}$$

$$x_3 = \theta. \tag{35}$$

In the first case, the ideal sliding dynamics can be described by the following set of equations:

$$\dot{x}_{2} = -\frac{ke}{1-k^{2}} + \frac{k\omega_{1}x_{3}}{1-k^{2}} + \frac{\omega_{2}x_{4}}{1-k^{2}} + \frac{\omega_{21}-k\omega_{1}}{1-k^{2}} \\ \cdot \frac{\omega_{1}x_{3}+k\omega_{2}x_{4}-e}{\omega_{1}-k\omega_{21}}$$
(36)

$$\dot{x}_3 = \omega_1 x_1 - (\omega_1 x_1 + \omega_{21} x_2) \frac{\omega_1 x_3 + k \omega_2 x_4 - e}{(\omega_1 - k \omega_{21}) x_3} \quad (37)$$

$$\dot{x}_4 = -\omega_2 x_2 - \omega_R x_4. \tag{38}$$

The asymptotic behavior of (36)–(38) can be represented by a stable equilibrium point whose coordinates are the following:

$$x_1^* = \theta, \ x_2^* = \frac{\sqrt{\omega_R e \theta}}{\omega_2},$$

$$x_3^* = \frac{e}{\omega_1} + \frac{\omega_2}{\omega_{21}} \sqrt{\frac{e \theta}{\omega_R}}, \ x_4^* = -\sqrt{\frac{e \theta}{\omega_R}}.$$
 (39)

The local stability conditions for this equilibrium point can be summarized as follows:

- 1) for $k \ge 0$, the system will be stable if $k\omega_{21} \omega_1 < 0$; and
- 2) for k < 0, the system is stable for values of x_4^* near zero, becoming unstable for large values of $|x_4^*|$.



Fig. 2. Output voltage response to input and load variations (Figs. 3 and 4) when the sliding mode uses the surface $2 \cdot 10^{-5} \ dv_{Co}/dt + v_{Co} + 5 = 0$.

The corresponding control law will be expressed as follows:

$$u = 1 \qquad \text{if } x_1 - \theta < 0 \tag{40}$$

$$u = 0 \qquad \text{if } x_1 - \theta > 0. \tag{41}$$

In the second case, the discontinuity surface $x_2 = \theta$ yields an ideal sliding dynamics containing a stable equilibrium point, provided that the following condition is satisfied:

$$\frac{ke}{\omega_2} < x_4^* < 0 \tag{42}$$

or, equivalently,

$$kV_g \sqrt{\frac{L_2}{L_1}} < v_{Co}^* < 0 \tag{43}$$

which implies that k must be negative.

On the other hand, if we consider the sliding surface $x_3 = \theta$, we obtain an unstable equilibrium point.

Although the sliding surface $x_4 = \theta$ does not satisfy the transversality condition, we can take the following surface:

$$\frac{dx_4}{dt} + \delta x_4 = \theta \tag{44}$$

which satisfies the transversality condition and results in the equivalent control [see (45), shown at the bottom of the page].

The corresponding ideal dynamics have a stable equilibrium point provided that condition (42) is accomplished and δ is positive.

In Fig. 2, a SIMPLIS simulation [25]–[27] shows the output voltage behavior when the input voltage and the load resistance exhibit simultaneously the pulsating characteristics depicted in Figs. 3 and 4, respectively. The converter parameters are $L_1 = L_2 = 1$ mH, k = -0.9, $C_1 = 1 \mu$ F, and $C_o = 20 \mu$ F.

$$u_{\rm eq} = \frac{k\omega_2\omega_1 x_3 + \omega_2^2 x_4 - k\omega_2 e + (1 - k^2)(\delta - \omega_R)(\omega_2 x_2 + \omega_R x_4)}{(k\omega_1 - \omega_{21})\omega_2 x_3}$$
(45)



Fig. 3. Input voltage characteristics.



Fig. 4. Pulsating characteristics of the load.



Fig. 5. Sliding surface behavior corresponding to Fig. 2.

The comparator hysteresis has been adjusted to provide a switching frequency around 50 kHz. The sliding surface is similar to a linear compensating network of PD type. The analogy suggests that the inclusion of an integral term in the surface would eliminate the steady-state error [33].

Fig. 5 illustrates the time-domain behavior of the sliding surface. There is loss of sliding regime at t = 2 ms corresponding to the first discontinuity of the input voltage and at instants corresponding to discontinuities in the output load. The loss of sliding regime is reflected at the same instants in the transient-state of the output voltage depicted in Fig. 2. However, if the input voltage has soft transitions between levels as shown in Fig. 6, there is a significant improvement of the transient



Fig. 6. Input voltage with soft transitions.



Fig. 7. Output voltage response to input and load variations (Figs. 4 and 6) when the sliding mode uses the same surface as in Fig. 2.



Fig. 8. Sliding surface behavior corresponding to Fig. 7.

behavior of the output voltage and no loss of sliding regime as depicted in Figs. 7 and 8, respectively.

B. Discontinuity Surfaces for Ideal Line Regulation

We have to choose those surfaces leading to sliding dynamics with equilibrium points in which the coordinate of the output variable x_4^* does not depend on e.

From (28) and (29), we conclude that the surfaces resulting in ideal line regulation will be of the form

$$\beta x_2 + \delta x_4 = \theta. \tag{46}$$



Fig. 9. Output voltage response to the input variations depicted in Fig. 3 when the sliding control uses the surface $i_{L2} - 0.2v_{Co} - 2 = 0$.

The ideal sliding dynamics will be described as

$$\dot{x}_{1} = \frac{e - \omega_{1}x_{3} - k\omega_{2}x_{4}}{1 - k^{2}} + \frac{\omega_{1} - k\omega_{21}}{1 - k^{2}} \\ \cdot \frac{-\beta(k\omega_{1}x_{3} + \omega_{2}x_{4} - ke) + \delta(\omega_{2}x_{2} + \omega_{R}x_{4})(1 - k^{2})}{\beta(\omega_{21} - k\omega_{1})}$$
(47)

$$\dot{x}_2 = \frac{\delta}{\beta} \left(\omega_2 x_2 + \omega_R x_4 \right) \tag{48}$$

$$\dot{x}_{3} = \omega_{1}x_{1} - (\omega_{1}x_{1} + \omega_{21}x_{2}) \\ \cdot \frac{-\beta(k\omega_{1}x_{3} + \omega_{2}x_{4} - ke) + \delta(\omega_{2}x_{2} + \omega_{R}x_{4})(1 - k^{2})}{\beta(\omega_{21} - k\omega_{1})x_{3}}$$
(49)

$$\dot{x}_4 = -\omega_2 \left(\frac{\theta}{\beta} - \frac{\delta}{\beta} x_4\right) - \omega_R x_4.$$
(50)

Since (50) does not depend on x_2 nor on x_3 , $x_4(t)$ will be given by

$$x_4(t) = K_1 e^{-(\omega_R - \omega_2 \delta/\beta)t} - \frac{\omega_2 \frac{\theta}{\beta}}{\omega_R - \frac{\omega_2 \delta}{\beta}}$$
(51)

Δ

which exhibits an asymptotically stable equilibrium point in $x_4^* = -[\theta \omega_2/(\beta \omega_R - \delta \omega_2)]$ provided that $\omega_R > \omega_2 \delta/\beta$.

Similarly, from (48) $x_2(t)$ will be expressed as

$$x_2(t) = K_2 e^{-(\omega_R - \omega_2 \delta/\beta)t} + \frac{\theta \omega_R}{\beta \omega_R - \delta \omega_2}$$
(52)

which reaches the stable equilibrium point $x_2^* = \theta \omega_R / (\beta \omega_R - \delta \omega_2)$.

Equations (47) and (49), in turn, result in an equilibrium point

$$\left(\frac{\omega_R x_4^{*2}}{e}, \frac{e}{\omega_1} - \frac{\omega_2}{\omega_{21}} x_4^*\right)$$

which is stable if (42) is satisfied.

In Fig. 9, a SIMPLIS simulation illustrates the insensitivity in steady state of the output voltage-to-input voltage variations. The converter parameters are $L_1 = L_2 = 1$ mH, k = -0.9, $C_1 = 1 \mu$ F, $C_o = 20 \mu$ F, $R_o = 5 \Omega$, and the input voltage has the time-varying behavior depicted in Fig. 3.

C. Ideal Load Regulation

Considering the surface $\alpha x_1 + \beta x_2 = 0$, the following ideal dynamics are obtained:

$$\dot{x}_{2} = \frac{-ke + k\omega_{1}x_{3} + \omega_{2}x_{4}}{1 - k^{2}} + \frac{\omega_{21} - k\omega_{1}}{1 - k^{2}} \\ \cdot \frac{\omega_{1}x_{3} + k\omega_{2}x_{4} - e - \beta(k\omega_{1}x_{3} + \omega_{2}x_{4} - ke)}{(\omega_{1} - k\omega_{21}) + \beta(\omega_{21} - k\omega_{1})}$$
(53)

$$\cdot \frac{\omega_1 x_2 + (\beta \omega_1 - \omega_{21})}{(\omega_1 - k\omega_{21}) + \beta(\omega_{21} - k\omega_{1})} (54)$$

$$\dot{x}_4 = -\omega_2 x_2 - \omega_R x_4 \tag{55}$$

where we have considered $\alpha = 1$ without loss of generality. The coordinates of the equilibrium point are

$$x_{1}^{*} = \frac{\beta \omega_{R}}{\omega_{2}} x_{4}^{*}, \ x_{2}^{*} = -\frac{\omega_{R}}{\omega_{2}} x_{4}^{*},$$
$$x_{3}^{*} = \frac{e}{\omega_{1}} - \frac{\omega_{2}}{\omega_{21}} x_{4}^{*}, \ x_{4}^{*} = \frac{\beta e}{\omega_{2}}.$$
(56)

From (32) and (56) we derive

$$\beta < 0.$$
 (57)

Applying the Routh's criterion to the characteristic polynomial of the linearized equations around the equilibrium point, it can be shown that the system will be asymptotically stable if β satisfies the inequality

$$-\frac{\sqrt{L_2} - k\sqrt{L_1}}{\sqrt{L_1} - k\sqrt{L_2}} < \beta < 0 \qquad 0 \le k < 1.$$
(58)

Considering $\alpha \neq 1$, the stability condition becomes

$$-\frac{\sqrt{L_2} - k\sqrt{L_1}}{\sqrt{L_1} - k\sqrt{L_2}} < \frac{\beta}{\alpha} < 0.$$
(59)

Equivalently,

$$-\frac{L_2 - M}{L_1 - M} < \frac{m_2}{m_1} < 0.$$
(60)

Similarly, we can express the surface for ideal load regulation in terms of the physical variables

$$m_1 i_{L1} + m_2 i_{L2} = 0. (61)$$

The coordinate of the output voltage will be given by

$$v_{Co}^* = \frac{m_2}{m_1} V_g \tag{62}$$

which is independent of the output load R_o .

The sliding domain for the output voltage will be expressed as

$$|v_{Co}^*| < V_g$$
 if $\left|\frac{m_2}{m_1}\right| < 1$ (step-down behavior) (63)

$$|v_{Co}^*| > V_g$$
 if $\left|\frac{m_2}{m_1}\right| > 1$ (step-up behavior). (64)

From (61) and (62), we obtain

$$V_g i_{L1}^* = -v_{Co}^* i_{L1}^* \tag{65}$$



Fig. 10. Output voltage response to a pulsating load shown in Fig. 4 when the sliding surface is $i_{L1} - 0.42i_{L2} = 0$.



Fig. 11. Trajectories in the plane $i_{L2} - v_{Co}$ when the load has the pulsating behavior of Fig. 4.

or, equivalently,

$$Pdc_{in} = Pdc_o$$
 (66)

where Pdc_{in} and Pdc_o are input and output power, respectively.

As no internal losses are considered, equality (66) could be expected which verifies the validity of the former results.

By comparing (61) to the general sliding surface (10), we conclude that there are no external references in the feedback loop and, therefore, the circuit will behave as a self-oscillating regulator. The corresponding control law is given by

$$u = 0 \qquad \text{if } m_1 i_{L1} + m_2 i_{L2} > 0 \tag{67}$$

$$u = 1$$
 if $m_1 i_{L1} + m_2 i_{L2} < 0.$ (68)

As an example, for the set of parameters $L_1 = L_2 = 1$ mH, k = 0.9, $C_1 = C_o = 20 \ \mu\text{F}$, and $V_g = 12 \ \text{V}$, the converter provides $-5 \ \text{V}$ in the output, and exhibits the transient behavior depicted in the SIMPLIS simulation of Fig. 10 when the load has the pulsating characteristics represented in Fig. 4. The converter sliding behavior in the plane $i_{L2} - v_{Co}$ is illustrated in Fig. 11. As shown in Figs. 10 and 11, the output voltage recovers its steady-state value of $-5 \ \text{V}$ after the transient states that follow the load changes.

D. Sliding Surfaces for Hysteretic Current Control

Hysteretic current control has been proposed in the coupledinductor Ćuk converter [28] as an alternative to the constant frequency peak current control, in order to avoid the intrinsic instability for duty cycles larger than 0.5, and the output jittering caused by the magnetic coupling. The hysteretic current control implemented in [28] can also be described in terms of the sliding-mode approach developed in this paper. Assuming $m_1 = m_2 = 1$ and $m_3 = m_4 = 0$ in the general sliding surface (10), we obtain

$$S(x) = (i_{L1} + i_{L2}) - (i_{L1}^* + i_{L2}^*).$$
(69)

In terms of the normalized variables, the ideal sliding dynamics will be given by

$$\dot{x}_{2} = \frac{k\omega_{1}x_{3} + \omega_{2}x_{4} - ke}{1 - k^{2}} + \frac{\omega_{21} - k\omega_{1}}{1 - k^{2}} \\ \cdot \frac{\alpha(\omega_{1}x_{3} + k\omega_{2}x_{4} - e) - \beta(k\omega_{1}x_{3} + \omega_{2}x_{4} - ke)}{\alpha(\omega_{1} - k\omega_{21}) + \beta(\omega_{21} - k\omega_{1})}$$

$$\dot{x}_{3} = \omega_{1} \frac{\theta - \beta x_{2}}{\alpha} + \left(-\omega_{1} \frac{\theta - \beta x_{2}}{\alpha} - \omega_{21}x_{2}\right) \\ \alpha(\omega_{1}x_{3} + k\omega_{2}x_{4} - e) - \beta(k\omega_{1}x_{3} + \omega_{2}x_{4} - ke)$$
(70)

$$\frac{\alpha(\omega_1 - k\omega_{21}) + \beta(\omega_{21} - k\omega_1)}{\alpha(71)}$$

$$\dot{x}_4 = -\omega_2 x_2 - \omega_R x_4. \tag{72}$$

Equations (70)–(72) have an equilibrium point in

$$x_{1}^{*} = \frac{\theta - \beta x_{4}^{*}}{\alpha}, \quad x_{2}^{*} = -\frac{\omega_{R}}{\omega_{2}} x_{4}^{*}, \quad x_{3}^{*} = \frac{e}{\omega_{1}} - \frac{\omega_{2}}{\omega_{21}} x_{4}^{*},$$
$$x_{4}^{*} = \frac{\frac{\beta \omega_{R}}{\omega_{2}} - \sqrt{\left(\frac{\beta \omega_{R}}{\omega_{2}}\right)^{2} - \frac{4\theta \alpha \omega_{R}}{e}}}{\frac{2\alpha \omega_{R}}{e}}.$$
(73)

The analysis of the linearized sliding dynamics around the equilibrium point shows that we can always find a subinterval $x_{4b} < x_4^* < 0$ where the system is stable, x_{4b} being finite and depending on the converter parameters.

The control law in this case is given by

$$u = 0 \qquad \text{if } i_S > i_{L1}^* + i_{L2}^* \tag{74}$$

$$\iota = 1 \qquad \text{if } i_S < i_{L1}^* + i_{L2}^* \tag{75}$$

where i_S represents the switch current defined as

$$i_S = i_1 + i_2.$$
 (76)

The implementation of this control law leads straightforwardly to the hysteretic current-controlled Ćuk converter presented in [28].

IV. CONCLUSIONS

Analytic solutions have been derived for the dynamic behavior of a bidirectional coupled-inductor Ćuk converter operating in sliding mode. The method of the equivalent control has been successfully applied to describe the converter ideal sliding dynamics under different discontinuity surfaces. Insensitivity in steady state to output-voltage variations can be obtained for certain values of the output voltage if the discontinuity surface is a linear combination of output voltage and output current, and the magnetic coupling between inductors is negative.

It has been demonstrated that the sensitivity of the output voltage to load variations can be reduced to zero in steady state if the regulator operates in a self-oscillating mode and the sliding surface is a linear combination with opposite signs of input and output currents. The inclusion of an external reference in the previous surface and the assumption of equal coefficients with equal signs show that the hysteretic current control can be described as a particular case of the sliding mode control.

Another important conclusion is that the converter can be controlled like a buck switching cell using a linear combination of the output voltage error and its derivative and minimum size of energy transfer capacitor.

The general sliding surface analyzed in this paper can also be applied in the control of other four-order structures [29]–[32]. Further research contemplates the implementation of the sliding controller by means of analog ASIC technology.

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